

Remark 2.4

In the case of continuous motions, from the material derivative of an integral formula (1.2.5) and the continuity equation (2.7) we get

$$\boxed{\frac{d}{dt} \int_D (\rho F) dV} = \int_D \left[ \frac{d}{dt} (\rho F) + \rho F \operatorname{div} \vec{v} \right] dv =$$

$$= \int_D \left[ \rho \frac{dF}{dt} + F \left( \frac{d\rho}{dt} + \rho \operatorname{div} \vec{v} \right) \right] dV = \boxed{\int_D \rho \frac{dF}{dt} dv} \quad (2.10)$$

We will use this formula when deriving the balance equations.

Definition: mass flux

Let's calculate the quantity of material through a surface  $S$  in an interval of time equal to 1 (see Figure 2.1). Let  $\Delta a$  be a surface element; the quantity of mass through  $\Delta a$  in  $\Delta t = 1$  is contained in the cylinder with generator  $\equiv |\vec{v}|$ ,

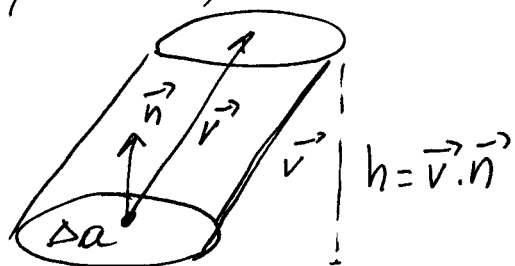


Figure 2.1

as  $|\vec{v}| =$  the distance per unit time. With sign convention that the flux is positive when  $\vec{v}$  and  $\vec{n}$

point in the same direction, or negative otherwise, we obtain for the flux of mass the formula

$$\phi = \int_S \rho \vec{v} \cdot \vec{n} da \quad (2.11)$$

as  $dm = \rho dV$ , and  $\Delta V = |\vec{v} \cdot \vec{n}| \Delta a$ .

In fact, in this way we can calculate the flux of any variable having a density  $f$ ; we get

$$\phi = \int_S f \vec{v} \cdot \vec{n} da \quad (2.12)$$

### 2.3. The Principle of the variation of momentum. FMZ 5

The momentum of a continuous system  $P$  with support  $D$  is

$$\vec{H} = \int_P \vec{v} dm = \int_D \rho \vec{v} dV. \quad (2.13)$$

Postulate: a material system  $M$  moves such that at any time  $t$ , <sup>and for any subsystem  $P$</sup>  the derivative of the momentum with respect to time is equal to the resultant of the external forces acting on  $P$ ,

$$\frac{d\vec{H}}{dt} = \vec{R}. \quad (2.14)$$

For a continuous medium we need to consider both contact forces of resultant  $\vec{R}^c$ ,

$$\vec{R}^c = \int_{\partial D} \vec{t} da \quad (2.15)$$

and momentum

$$\vec{M}^c = \int_{\partial D} \vec{x} \times \vec{t} da \quad (2.16)$$

and distance forces like gravitational attraction, of density  $\vec{f}$ ; their resultant force and resultant momentum are

$$\vec{R}^d = \int_P \vec{f} dm = \int_D \rho \vec{f} dV, \quad (2.17)$$

and

$$\vec{M}^d = \int_P \vec{x} \times \vec{f} dm = \int_D \rho \vec{x} \times \vec{f} dV, \quad (2.18)$$

respectively.

From (2.15) - (2.18) and (2.14) we get that

$$\frac{d}{dt} \int_D \rho \vec{v} dV = \int_{\partial D} \vec{t} da + \int_D \rho \vec{f} dV, \quad (\forall) D \subset \mathcal{D}, \quad (2.19)$$

The above formula is valid for both continuous and discontinuous motions. For continuous motions only, if we take into account (2.10), we get the momentum equation under the form

$$\int_D \rho \vec{a} dV = \int_{\partial D} \vec{t} da + \int_D \rho \vec{f} dV \quad (2.20)$$

### 2.4. The Principle of Variation of the Kinetic Momentum

The kinetic momentum of any part  $P$  of a continuous material system  $\mathcal{M}$  can be defined as

$$\vec{K} = \int_P \vec{x} \times \vec{v} dm = \int_D \rho \vec{x} \times \vec{v} dV \quad (2.21)$$

#### The Principle of variation of the kinetic momentum:

A material system  $\mathcal{M}$  moves such that, at any time  $t$  and for any sub-system  $P$  of  $\mathcal{M}$ , the time derivative of the kinetic momentum equals the resultant momentum of external forces acting on  $P$ .

Analytically, we can write the above principle as

$$\frac{d}{dt} \int_D \rho \vec{x} \times \vec{v} dV = \int_{\partial D} \vec{x} \times \vec{t} da + \int_D \rho \vec{x} \times \vec{f} dV \quad (2.22)$$

and for continuous motions, using the formula (2.10), we get that

$$\int_D \vec{x} \times \vec{a} \rho dV = \int_{\partial D} \vec{x} \times \vec{t} da + \int_D \rho \vec{x} \times \vec{f} da \quad (2.23)$$

## 2.5. Stress Tensor. Cauchy's equations.

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### Cauchy's Lemma \*)

If  $\vec{t}$  is continuous w.r.t.  $\vec{x}$ , then at any  $\vec{x} \in \mathcal{D}$  we have

$$\vec{t}(\vec{x}, \vec{n}) = -\vec{t}(\vec{x}, -\vec{n}) \quad (2.24)$$

#### Proof

Consider a cylinder  $D_h$ , centered at  $\vec{x}$ , such that it reduces to the disc  $\Sigma$  when its height  $h \rightarrow 0$  (see Figure 2.2). From (2.20) we have

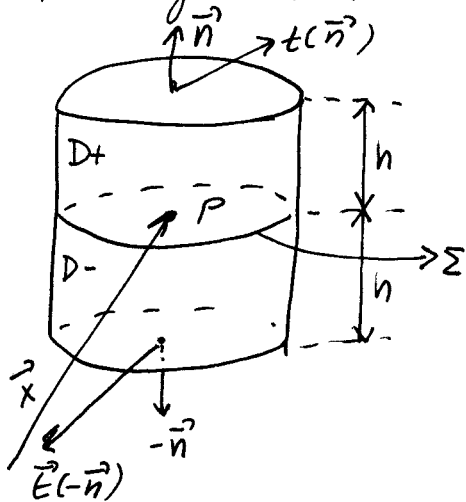


Figure 2.2.

$$\int_{D_+} \rho \vec{a} \, dv = \int_{\partial D_+} \vec{t} \, da + \int_{D_+} \rho \vec{f} \, dv$$

and the volume integrals  $\int_{D_+}$  as well as the integrals on the surface of  $D_+$  (the external surface) approach zero as  $h \rightarrow 0$ . We are left with

$$\int_{\Sigma} [\vec{t}(\vec{x}, \vec{n}) + \vec{t}(\vec{x}, -\vec{n})] \, da = 0$$

and from the Fundamental Lemma we get that  $\vec{t}(\vec{x}, \vec{n}) + \vec{t}(\vec{x}, -\vec{n}) = 0$  q.e.d.

### Cauchy's Theorem

If  $\vec{t}$  is a continuous function on  $\mathcal{D}$  then there exists a tensor  $\vec{T}(\vec{x})$ , defined on  $\mathcal{D}$ , such that at any point  $P(\vec{x})$  we have

$$\vec{t}(\vec{x}, \vec{n}) = \vec{T}(\vec{x}) \vec{n}, \quad (2.25)$$

or, on components,

$$t_i(\vec{x}, \vec{n}) = T_{ij} n_j, \quad i=1,2,3 \quad (2.26)$$

Remark 2.4. Cauchy's theorem shows that  $\vec{t}$  is a linear function of  $\vec{n}$ .

\*) We consider the current (Eulerian) configuration, at an instant time  $t$ , so we omit  $t$  from  $\vec{t}(\vec{x}, \vec{n}, t)$ .

Remark 2.5.  $\vec{T}(\vec{x})$  characterizes completely the stress state at  $P(\vec{x})$ . Indeed, if we know  $T(\vec{x})$  we can then calculate the stress vector  $\vec{T}$  for any oriented surface of normal  $\vec{n}$ .

Proof

We consider a tetrahedron centered at  $P$  and having three sides perpendicular to the coordinate axes (see Fig. 2.3). Denote by  $h$  the length of the perpendicular from  $P$  to the opposite side  $P_1P_2P_3$ . We observe that the volume of the tetrahedron approaches zero when  $h \rightarrow 0$ . This is not a particular

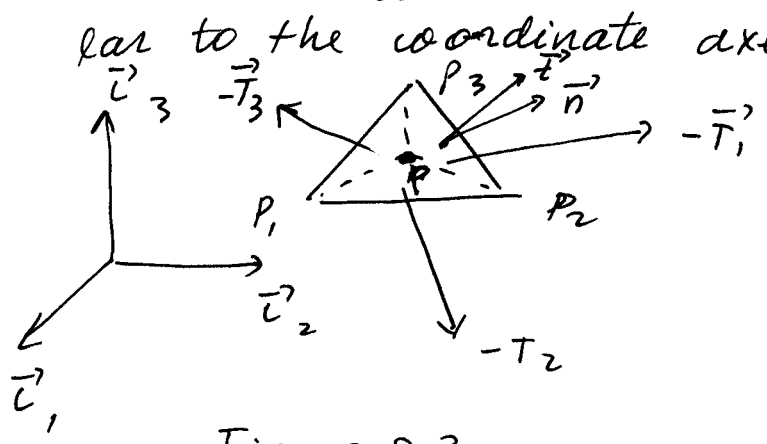


Figure 2.3.

choice since the dependence of  $\vec{T}$  on  $\vec{n}$  is unique. Denote by  $\vec{n}$  the normal to  $P_1P_2P_3$ , and by  $\Delta a_i, i=1,2,3$ , the areas of the sides  $PP_2P_3, PP_1P_3, PP_2P_1$ . We have denoted by  $\vec{T}_j(\vec{x}) = \vec{T}(\vec{x}, \vec{c}_j)$ . According to Cauchy's Lemma we denote by  $-\vec{T}_j$  the stress vector on the side  $PP_2P_3$ ; analogously we can define  $-\vec{T}_2$  and  $-\vec{T}_3$ . If  $\Delta a$  is the area of  $P_1P_2P_3$ , we have

$$\Delta a_i = n_i \Delta a, \quad i=1,2,3 \quad (2.27)$$

The volume of the tetrahedron is  $\frac{1}{3} h \Delta a$ . We apply the principle of the variation of momentum with  $D =$  the tetrahedron and we use the formula

$$\int_D \vec{F}(\vec{x}) dV = (\vec{F}(\vec{\xi}) + \vec{\varepsilon}) \int_D dV,$$

where  $\lim_{\mu(D) \rightarrow 0} \varepsilon = 0$ . We get, from (2.2.7),

$$\rho(\vec{x}) [\vec{a}(\vec{x}) - \vec{f}(\vec{x}) - \vec{\varepsilon}] \frac{1}{3} h \Delta a = [\vec{T}(\vec{x}, \vec{n}) + \vec{\varepsilon}_n] \Delta a - \sum_j [\vec{T}_j(\vec{x}) + \vec{\varepsilon}_j] \Delta a_j$$

with  $\lim_{h \rightarrow 0} (\epsilon, \epsilon_n, \epsilon_j) = 0$ .

Using (2.27), dividing by  $\Delta a$  and taking the limit  $h \rightarrow 0$  we get

$$\vec{t}(\vec{x}, \vec{n}) = T_j (\vec{x}) n_j \tag{2.28}$$

After projection of (2.28) on  $Ox_i$  we get (2.26)  $\square$

Remark 2.6.

The formula (2.26) is the proof that  $\overleftrightarrow{T}$  is a tensor. It is the stress tensor, the first tensor to be defined in sciences. The name "tensor" comes from "tension".

Remark 2.7

Let's project  $t(\vec{x}, \vec{n})$  along a direction of unit vector  $\vec{m}$ . We have that

$$\vec{m} \cdot \vec{t} = \vec{m} \cdot \overleftrightarrow{T} \vec{n} = T_{ij} m_i n_j \tag{2.29}$$

This formula helps us to show that if we change the basis from  $\{\vec{e}_1, \vec{e}_2, \vec{e}_3\}$  to another orthogonal basis  $\{\vec{e}'_1, \vec{e}'_2, \vec{e}'_3\}$ , the quantity  $\overleftrightarrow{T}$  changes according to the formula

$$\overleftrightarrow{T}' = Q \overleftrightarrow{T} Q^T \tag{2.30}$$

where  $Q$  is the change of base matrix. Equivalently,

$$T'_{ke} = Q_{ke} Q_{ij} T_{ij} \tag{2.30}'$$

If we take  $\vec{m} = \vec{n}$  we get the formula for the normal stress

$$N = \vec{n} \cdot \vec{t} = \vec{n} \cdot \overleftrightarrow{T} \vec{n} = T_{ij} n_i n_j \tag{2.31}$$

Question

Find those directions  $\vec{m}$  for which  $N$  takes extremal values.